

AMENDED CLAIMS

Claims 1-15 (Cancelled)

16.(New) A pneumatic fixing machine comprising:
a piston mean (30) slidably located inside a cylinder (33);
first valve means (5) movable between two extreme positions, an opening position (E), such that the first valve means are in fluidic communication with an inlet portion (33a) of said cylinder (33) for receiving a pressurized fluid and a closing position (D) where the first valve means are in communication with an external outlet;
second valve means (2) movable in response to operation of a trigger (7), the second valve means movable between an occlusion position (M) for fluidly connecting at least a first duct (9) and a base portion (5a) of the first valve means (5) for feeding the pressurized fluid therethrough, and, a passage position (L) for connecting to an external outlet;
third valve means (3) movable between a position of obstruction (N), to cut off flow between the first duct (9) and the first valve means (5), and, a crossing position (Q), which opens a flow connection between the duct (9) and the first valve means (5);
fourth valve means (4) movable in response to operation of the trigger (7), the fourth valve means movable between a block position (S), where flow is obstructed between the cylinder (33) and the third valve means (3), and, a transit position (T) such that flow is permitted between the cylinder (33) and the third valve means (3);
wherein in a first activation condition (A), the trigger (7) is partially pressed, positioning the second valve means (2) in the passage position (L), to allow the pressurized fluid to push the first valve means (5) into the opening position (E), actuating the piston mean (30);
and wherein in a second activation condition (B), the trigger (7) is further and completely pressed, moving the fourth valve means (4) into the transit position (T), to allow the pressurized fluid coming from the cylinder (33) to move the third valve means

(3) into the occlusion position (N), causing the return of the first valve means (5) to the closing position (D) and the consequent return of the piston mean (30) to the initial position (F).

17. (New) The pneumatic fixing machine according to claim 16 wherein the fourth valve means (4) comprise a lengthened and shaped element, slidably housed inside a first seat (12) which is complementary shaped and in fluid communication with the cylinder (30) through a second duct (16).

18. (New) The pneumatic fixing machine according to claim 17 wherein the fourth valve means (4) has a hollowed portion (14) for receiving a shaped free end (7a) of the trigger (7).

19. (New) The pneumatic machine according to claim 17 further comprising elastic means (13) for exerting a biasing force on the fourth valve means (4) to bias the fourth means towards the block position (S), in absence of operation of the trigger (7).

20. (New) The pneumatic machine according to claim 19 wherein the elastic means (13) comprise a compression spring received in the first seat (12) and on a prominence (4a) of the fourth valve means (4).

21. (New) The pneumatic machine according to claim 16 wherein the third valve means (3) comprise a shaped stem having a first end portion (3a) slidably housed in a second seat (15) and a second end portion (3b) slidably housed in a first room (10), the first end portion (3a) being shaped to occlude the exit of the first duct (9) inside the second seat (15) when the third valve means are in the occlusion position (N).

22. (New) The pneumatic machine according to claim 21 wherein the second end portion (3b) has an equivalent transversal section greater than an equivalent transversal section of the first end portion (3a).

23. (New) The pneumatic machine according to claim 17 further comprising a third duct (17) for fluid communication between the first seat (12) and the first room (10).

24. (New) The pneumatic machine according to claim 21 further comprising a third duct (17) for fluid communication between the first seat (12) and the first room (10).

25. (New) The pneumatic machine according to claim 23 wherein the fourth valve means \
(4) has a hollowed portion (4b) for fluid communication between the second duct (16) and the third duct (17) when the fourth valve means are in the transit position (T).

26. (New) The pneumatic machine according to claim 24 wherein the fourth valve means \
(4) has a hollowed portion (4b) for fluid communication between the second duct (16) and the third duct (17) when the fourth valve means are in the transit position (T).

27. (New) The pneumatic machine according to claim 21 wherein the base portion (5a) of the first valve means (5) is slidably housed in a second room (8) which is in fluid communication with the second seat (15).

28. (New) The pneumatic machine according to claim 16 wherein the first valve means (5) has a closing portion (5b) shaped to cut off pressurized fluid flow when the first valve means are in the closing condition (D), said closing portion (5b) having a smaller

equivalent transversal section than an equivalent transversal section of the base portion (5a).

29. (New) The pneumatic machine according to claim 21 wherein the first valve means (2) comprise a lengthened and shaped element, slidably housed inside a third seat (20) which has a first opening (23) for fluid communication with a third room (21) that it is in fluid communication with the feed of pressurized fluid, a second opening (24) intermediately located and connected to the first duct (9) and a third opening (22) for connecting with the external outlet.

30. (New) The pneumatic machine according to claim 29 wherein the first valve means (2) has an external portion (2a), fit to match the trigger (7), and a lock portion (2b), opposite to said external portion (2a) and fit for closing the first opening (23) of the third seat (20) when the first valve means are in the occlusion position (M).

31. (New) The pneumatic machine according to claim 16 wherein a geometric axis of the second valve means (2) and of the fourth valve means (4) are mutually perpendicular.

32. (New) The pneumatic machine according to claim 16 wherein a geometric axis of the first valve means (5) and of the third valve means (3) are nearly coincident and nearly parallel to an axis of the second valve means (2).